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Society of Petroleum Engineers

# SPE EnergyStream

## ● Live Webinar

### Extended Reach Drilling:

- OCTG Torque vs Tension: A Growing Concern for Well Integrity
- From Design to Execution: Advancing ERD with Real-Time Intelligence and Automation



**WEDNESDAY, 18 MARCH**  
1100 CDT (UTC-5)



**MODERATOR**

**MUHAMMAD NAVAID KHAN**  
ADNOC



**SPEAKER**

**WESLEY OTT**  
FERMATA CONNECTIONS



**SPEAKER**

**SHASHI TALYA**  
HALLIBURTON



**REGULATORY RESPONDENT**

**SEAN AVITT**  
RAILROAD COMMISSION  
OF TEXAS

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# Advancing Extended Reach Drilling From Design to Execution with Real-Time Intelligence and Automation

**Shashi Talya**  
March 18, 2026

**HALLIBURTON**

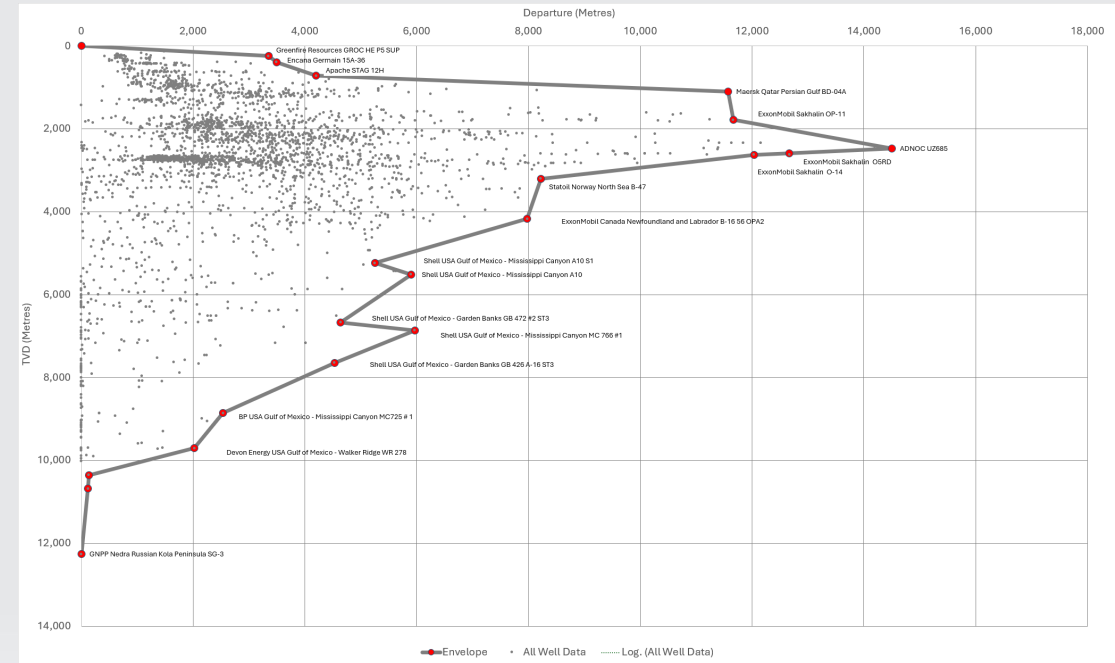
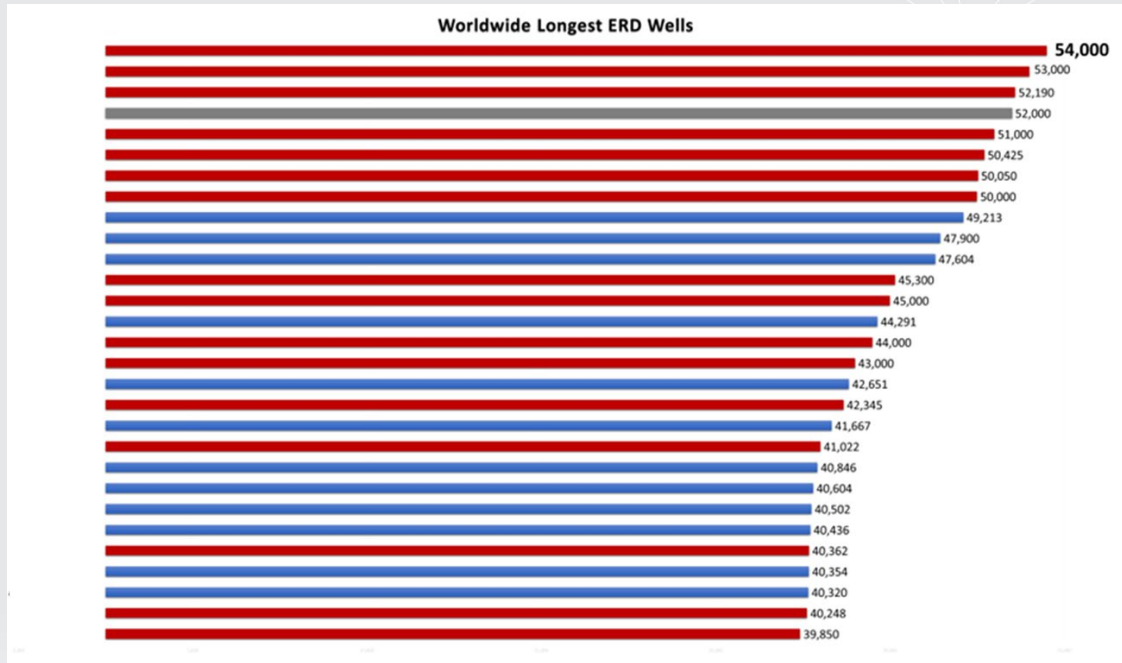
# Outline

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- Evolution of Extended Reach Drilling
- Key Challenges in ERD Execution
- Integrated Digital and Automation Solutions
- What's Next: Scaling Intelligence

# ERD History: 2019 – 2025



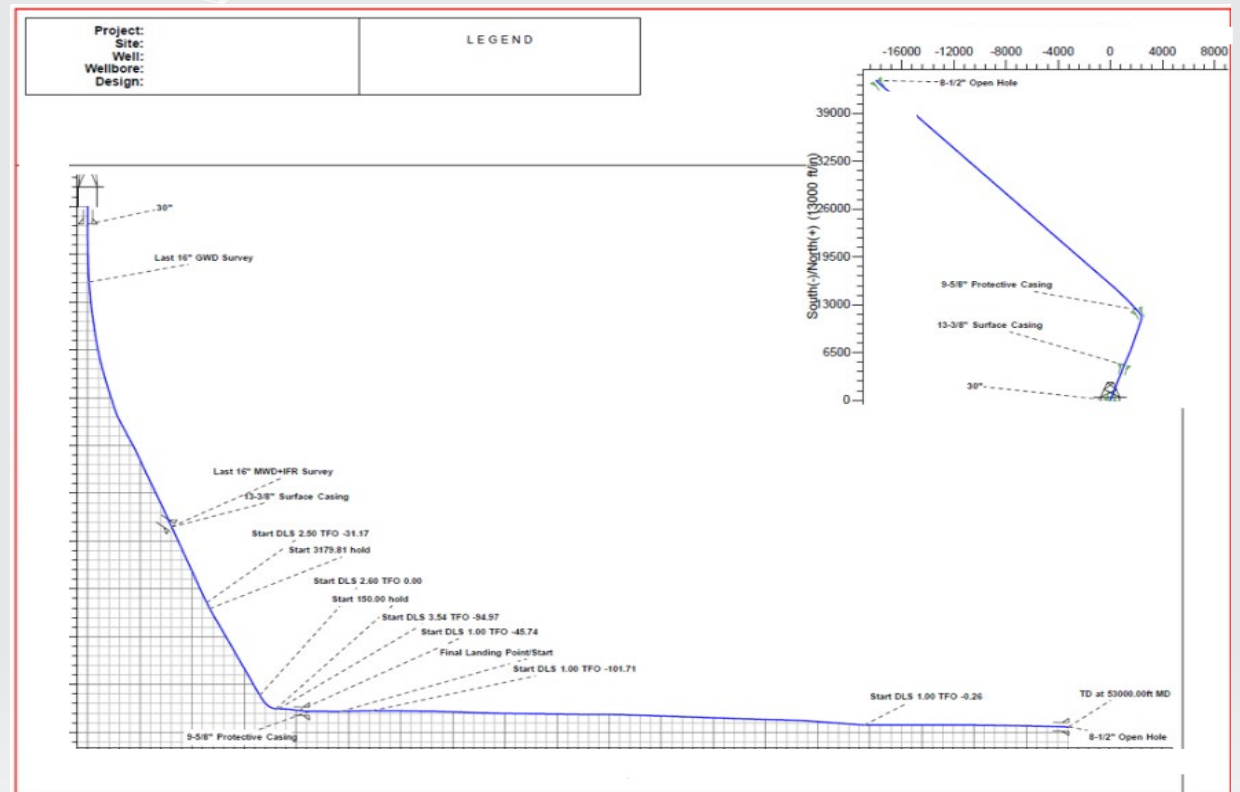
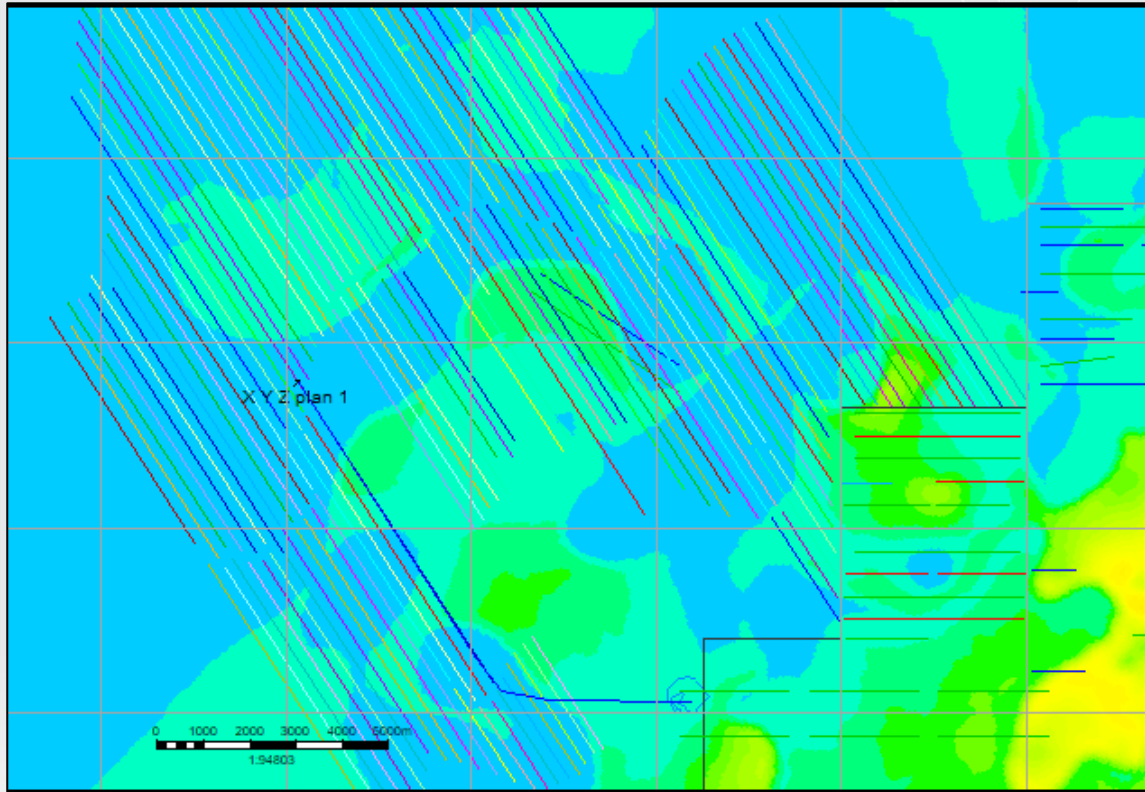
Source: MerlinERD

# Key Challenges in ERD Execution



Planned well spacing: 250 m

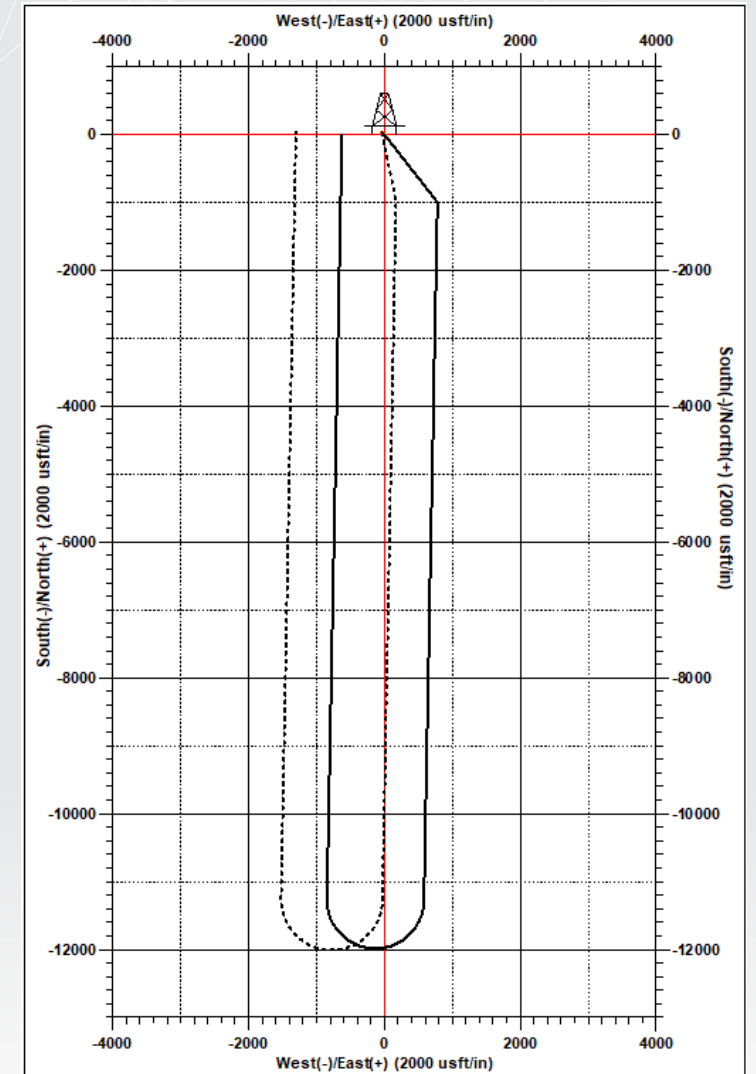
53,000 ft MD; Target reservoir: 6-10 ft thickness



# North America ERD: Regulatory & Operational Constraints




- Tight lease space
- Manage distance to lease lines, while maximizing lateral length
- Pad drilling density constraints
- Well spacing: 150-330 ft
- Directional control in the “U” section



# Challenges and Engineering Tradeoffs

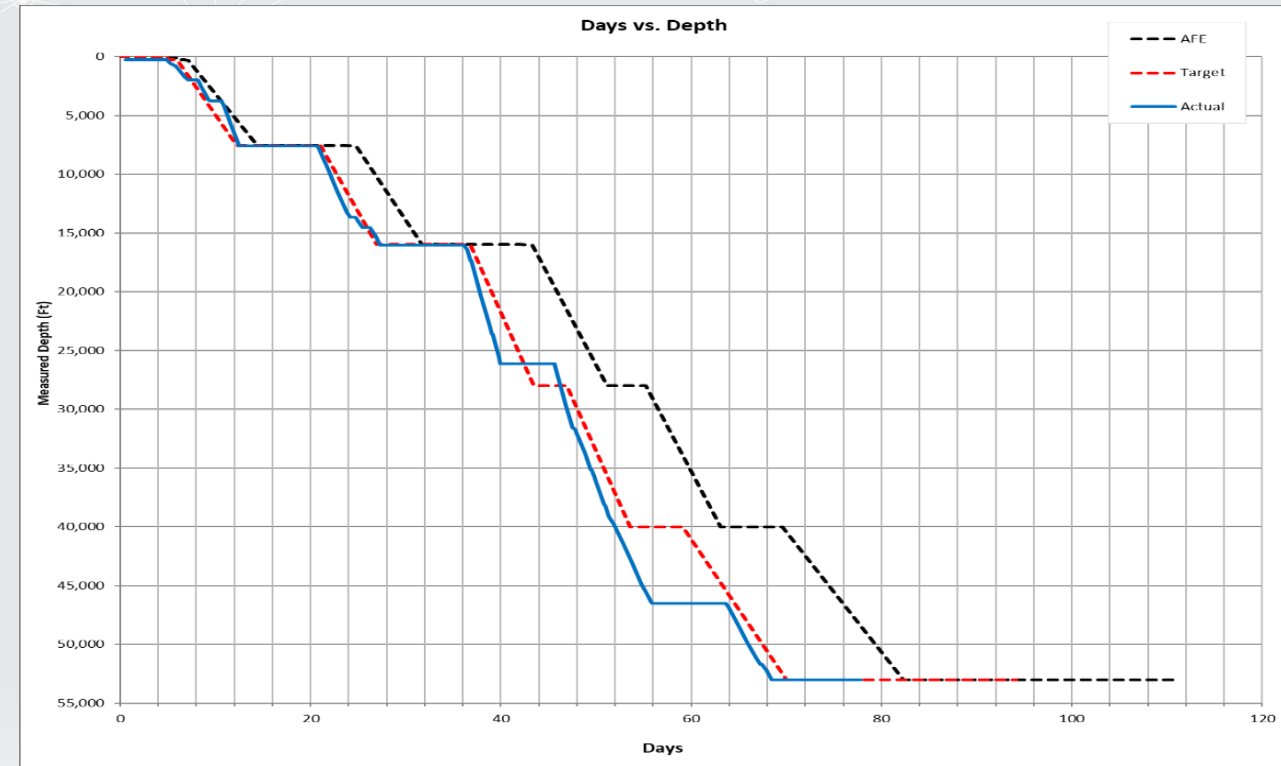


- Subsurface uncertainty
  - Survey uncertainty
  - Torque and drag management
  - Wellbore integrity
  - Tool reliability and durability
- 
- LWD sensors: Higher DOI (deep azimuthal resistivity) and near bit sensors
  - Downhole weight and torque optimization
  - Drive system: motor-assisted RSS or standalone RSS selection
  - Optimized BHA stabilization
  - Compact BHA design (fewer connections, fewer internal components)
  - Mud system optimization, friction reduction, and hole cleaning

# Drilling and Completing 53,000 ft Ultra-ERD Well



- Push-the-bit RSS with matched drill bit
- Advanced LWD and DAR tools, real-time formation evaluation and geosteering
- 100% reservoir contact achieved
- Delivered well 14 days ahead of schedule



SPE-227830-MS

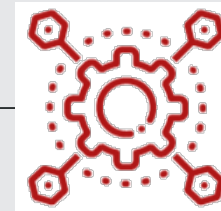
# Real-Time Execution: Integrated Control



**Drilling Rig Processes**  
*Drilling activity & Safeguards*

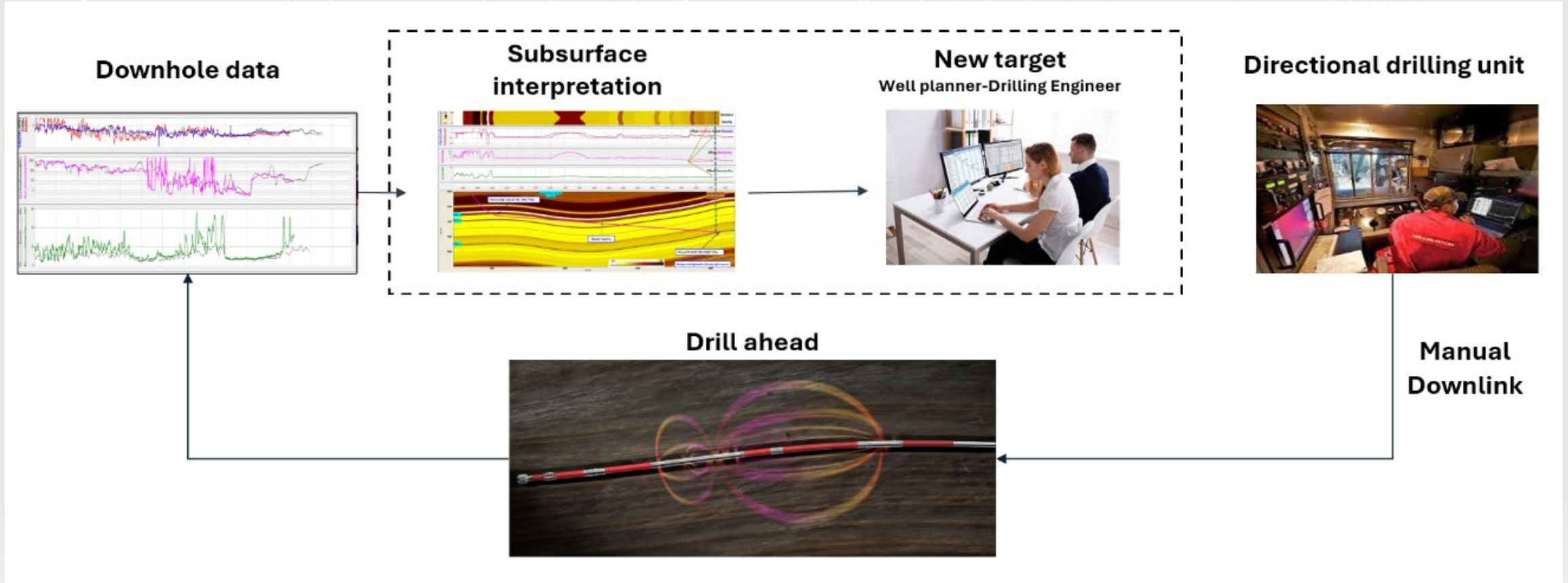
**Wellbore Integrity**  
*Rheology, Hydraulics and Optimization*

**Well Placement**  
*Trajectory and Sensing*

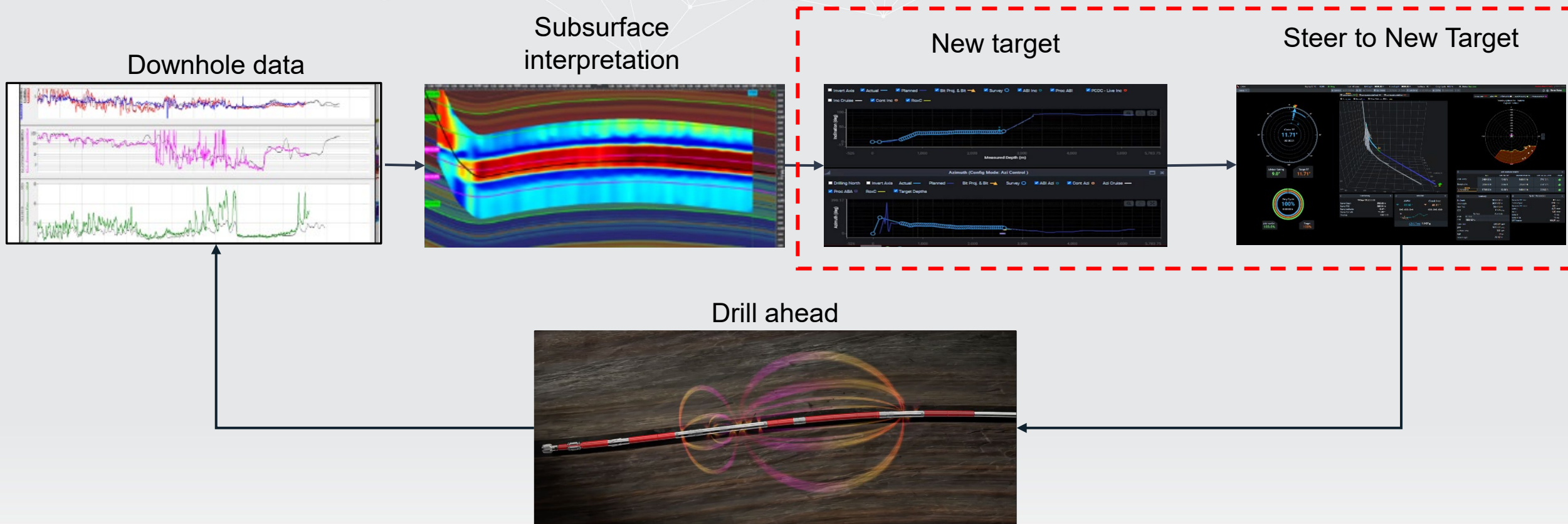


Integration of these 3 core components drives the degree of Automation

# Conventional Well Placement Workflow



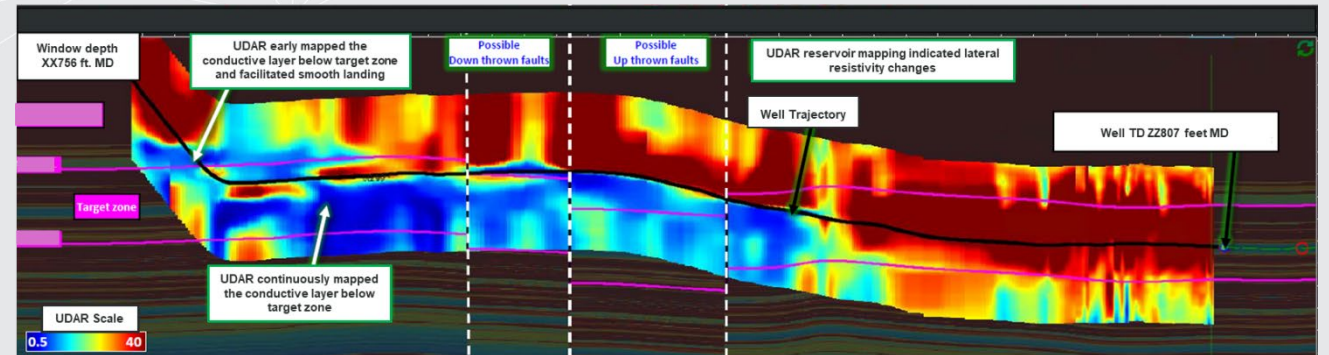
# Automated Target Placement



# Results



- 100% reservoir placement
- 19% ROP improvement
- >60% reduction in downlinks



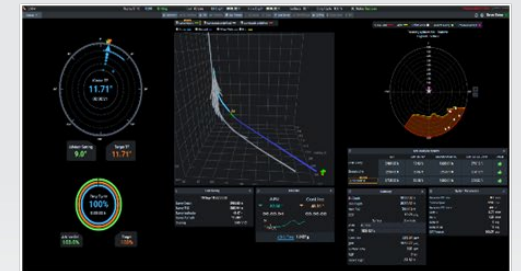
3D Geosteering



New targets and well path projection



Automated Well Placement

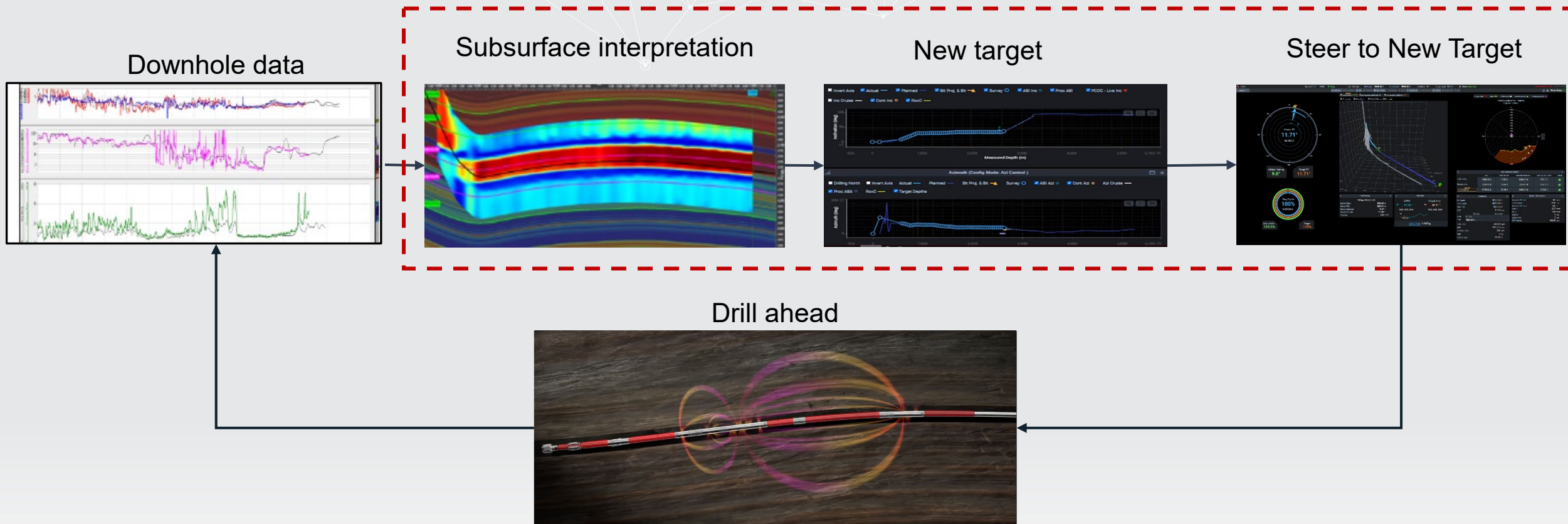


SPE-224743

# Geological Well Placement



## Automated Geosteering



# Key Takeaways and What's Next

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- Digital and automation are powering the next step change in ERD performance and consistency
- Stronger subsurface–drilling integration is key to maximizing reservoir exposure
- Managing torque and mechanical complexity becomes critical as ERD wells extend further
- Higher confidence in well positioning will come from improved surveying and modeling



**THANK YOU**

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# Extended Reach Drilling

OCTG Torque vs Tension A Growing Concern for Well Integrity

SPE-232499-MS

**Wesley L. Ott**  
March 18, 2026



# Extended Reach Drilling (ERD) / U-turn Laterals



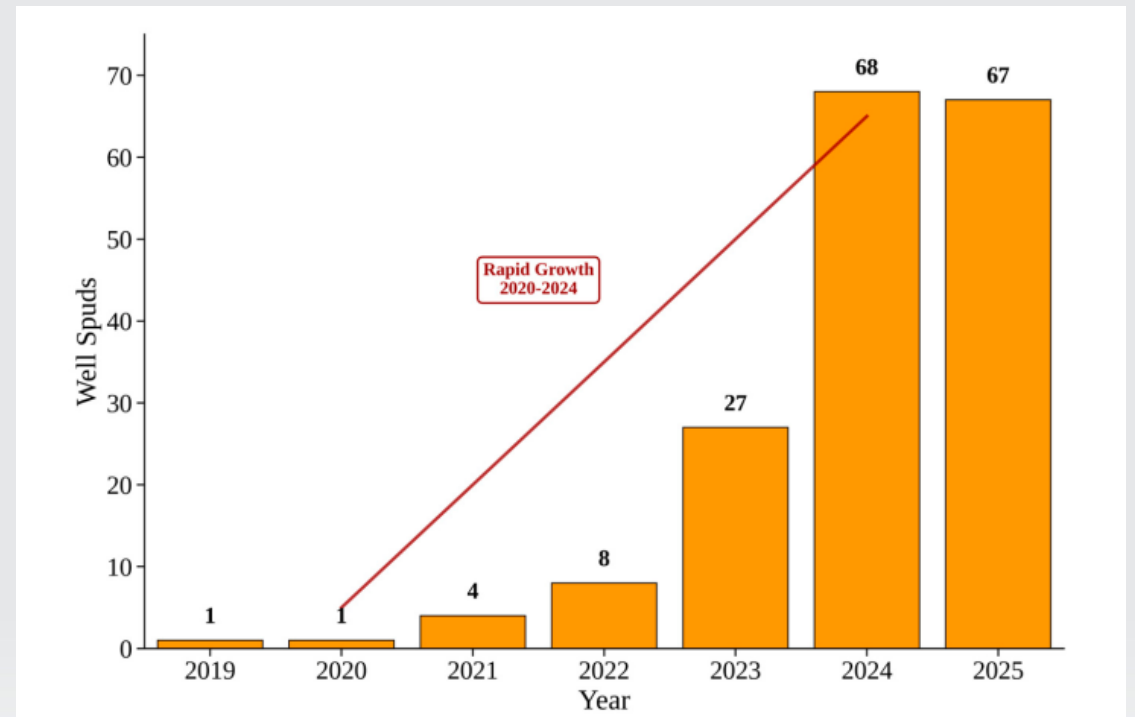
ERD is when  $HD > 2 \times TVD$  [1]

U-turn / horseshoe / hairpin / alternate shape / J-hook / Paperclip / Staple

## Technology Adoption Curve

- Feasibility
  - Shell / Delaware and Chesapeake / Eagleford
- Confidence inflection
- Scaled up adoption

## Annual U-lateral well spuds in the United States



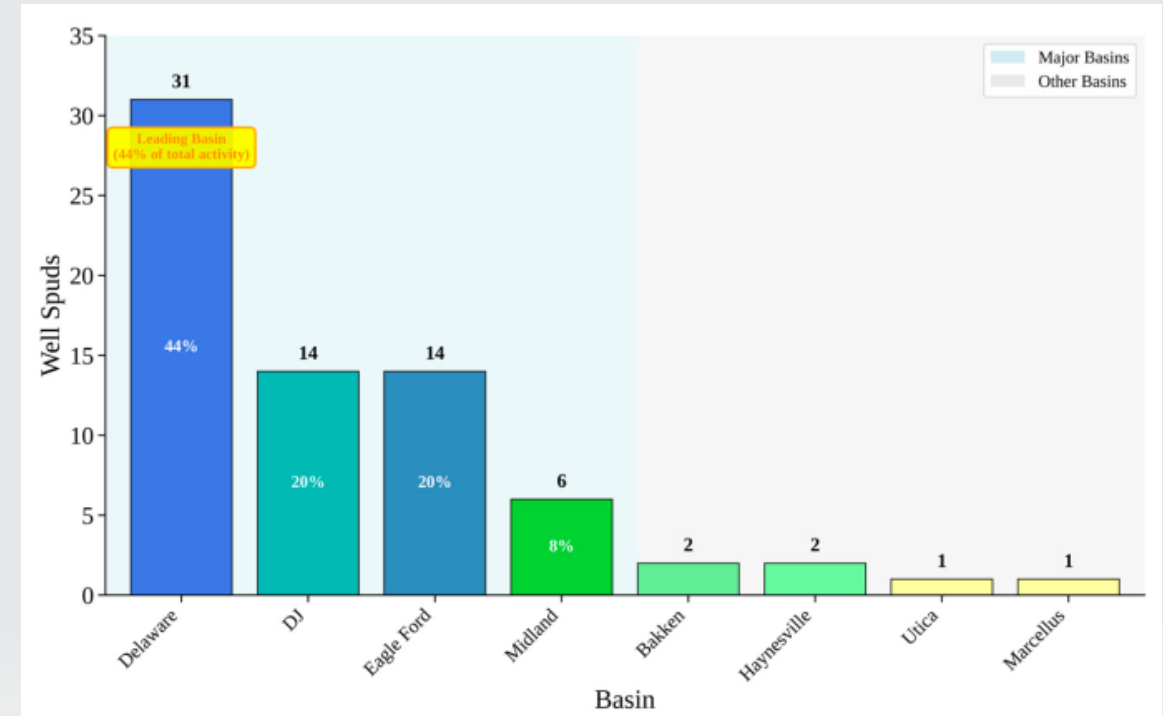
# ERW U-turn Laterals



U-lateral wells distribution by basin

## Primary Divers

1. Surface Lease Geometry / Stranded Acreage
2. Capital efficiency vs separate wells
  - 25% cost savings cradle to grave. [1,2]
3. Reduced surface footprint (regulatory)
  - 29.3% GHG reduction, 15.8% water reduction, 50% land disturbance reduction. [1,2]



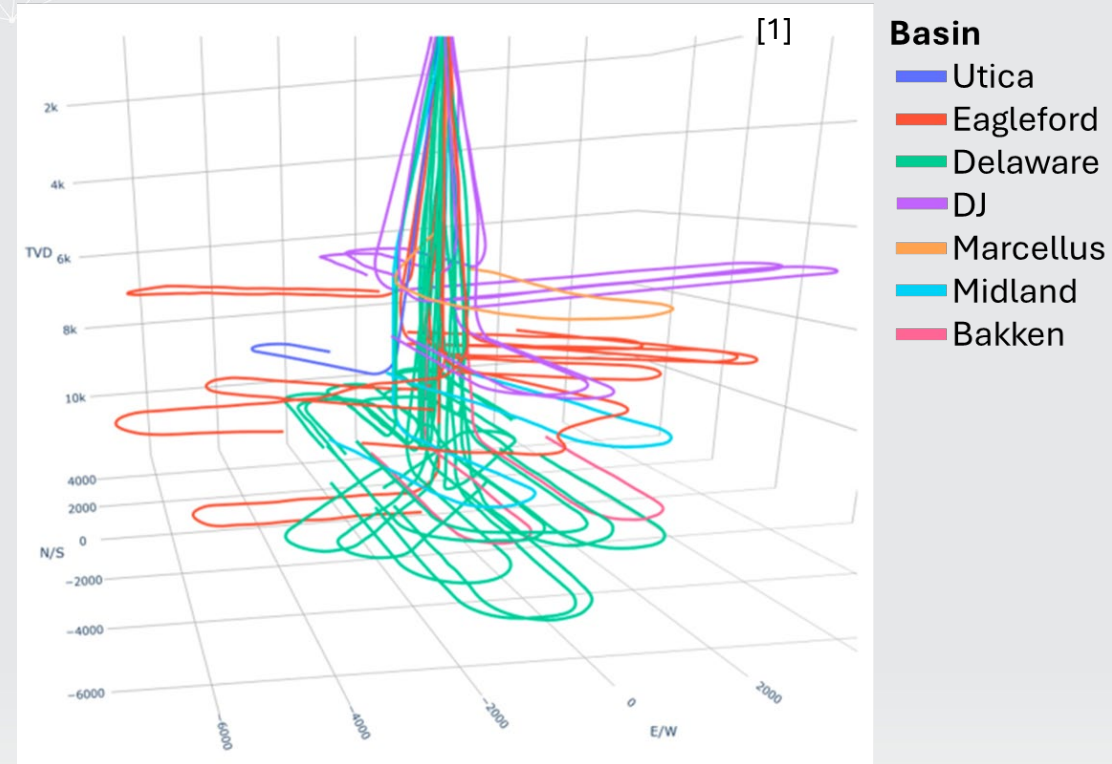
[1]

# Technical Challenges – Casing Lens

- High-curvature turn T&D + buckling sensitivity
  - Primary design constraint [2]
- Wellbore quality/tortuosity
  - Drives casing-running margins[3]

## Mitigation methods

1. Mud conditioning
2. Flotation
3. Rotation
4. Axial vibration
5. Swivels



# Casing Considerations



- Increased lateral lengths – increased T&D
  - Rotation mitigation
- Top-drive advancements
  - no longer the limiting factor

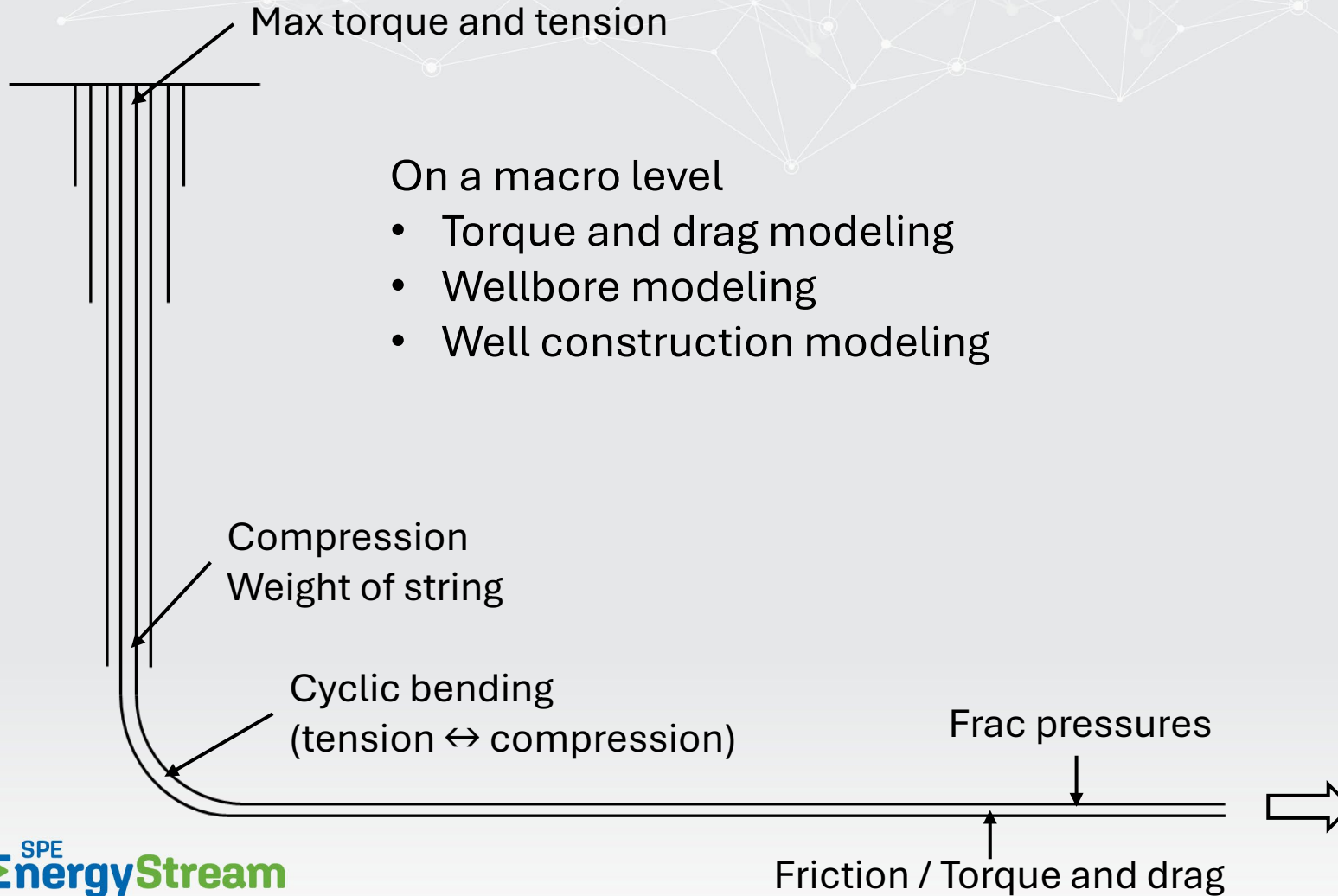
Next Weakest Link → Casing Connections

- Rotate through the bend (plural bends now)
- Clearance constraints (vs drill pipe)
- New combination of load conditions

## Split coupling field failure



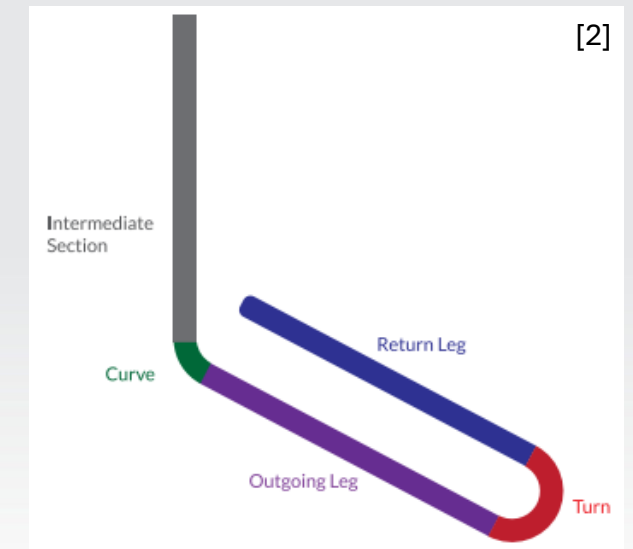
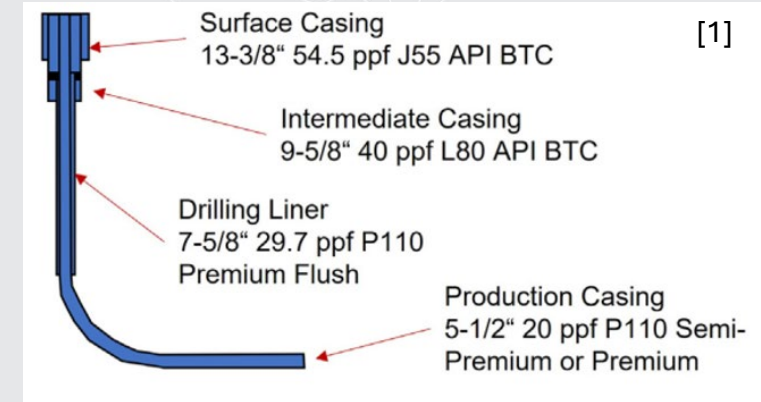
# Load State Along a Horizontal Well



On a macro level

- Torque and drag modeling
- Wellbore modeling
- Well construction modeling

## Midland Example

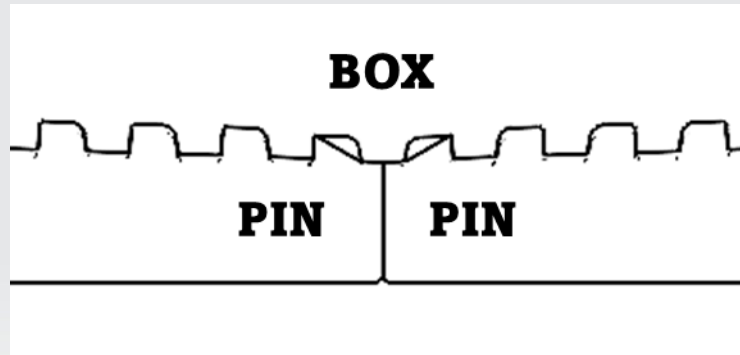


# Zoomed-In Load State

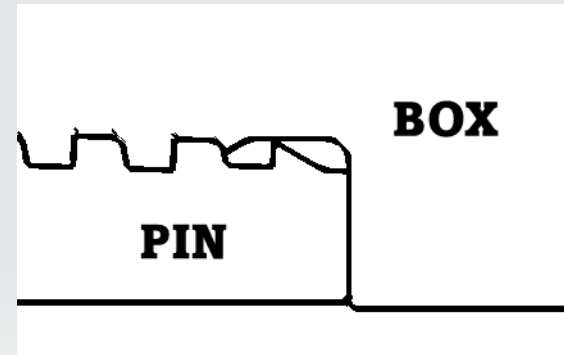
The load considerations are well understood on the macro-level modeling

Load considerations missed at the connection level

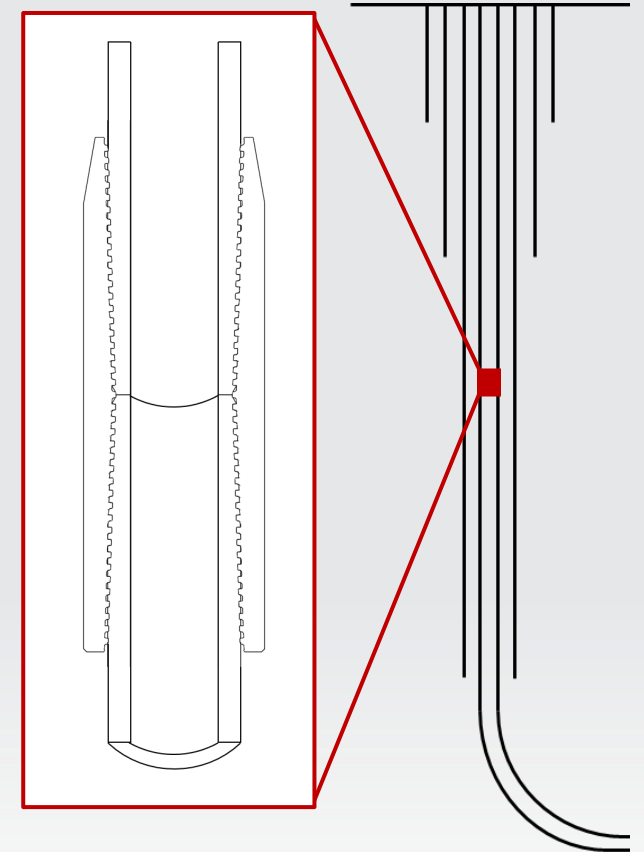
- Buttress compatible (BTC) is standard widely used connection
- Higher torque compatible options rely on a shoulder vs interference



BTC Pin-to-pin Shoulder



BTC Shouldered

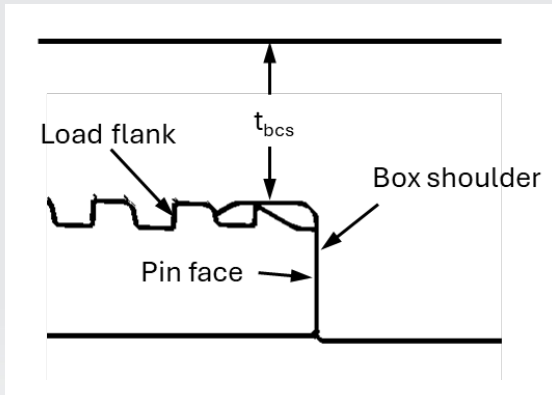


# Connection Load State

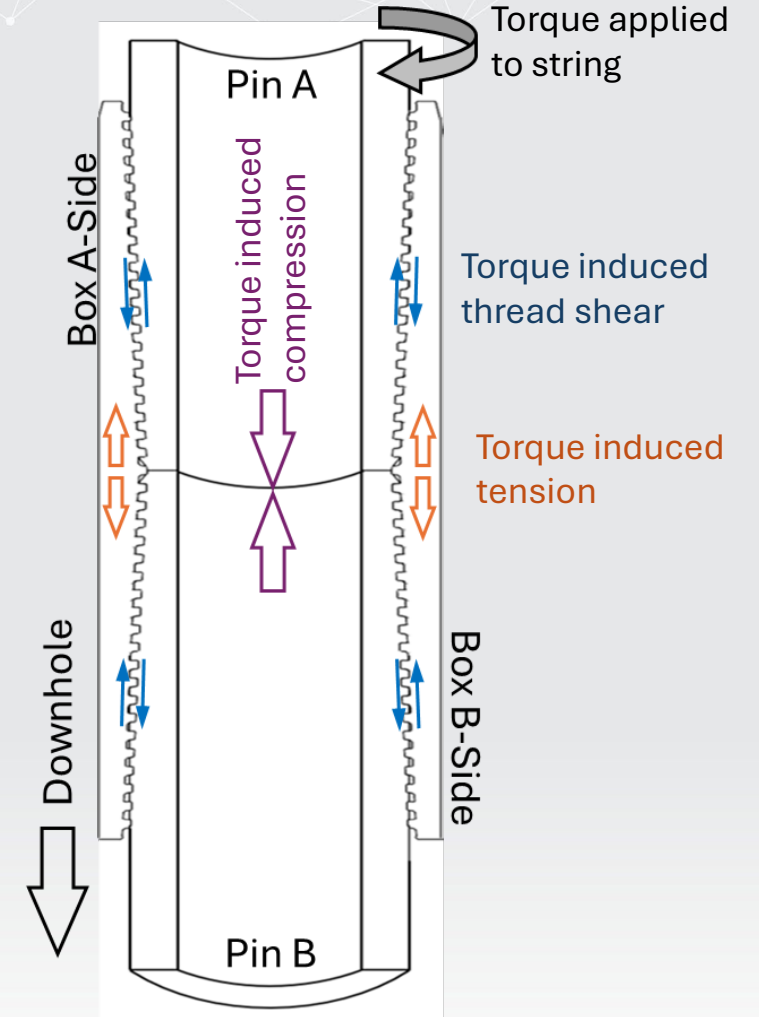
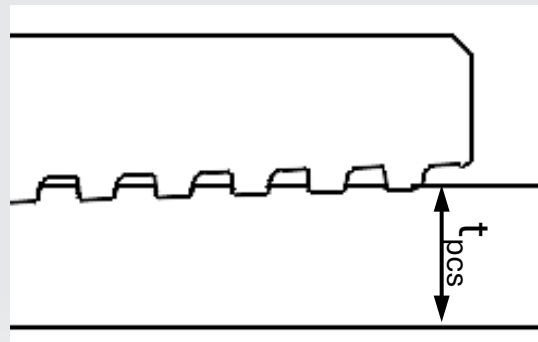
## Connection terminology

As a shouldered connection is rotated down-hole it will enact a tension load on the couplings critical cross-sectional area

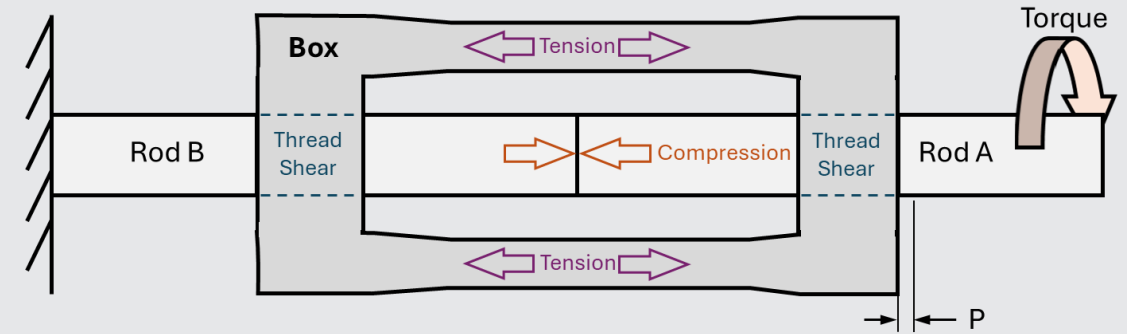
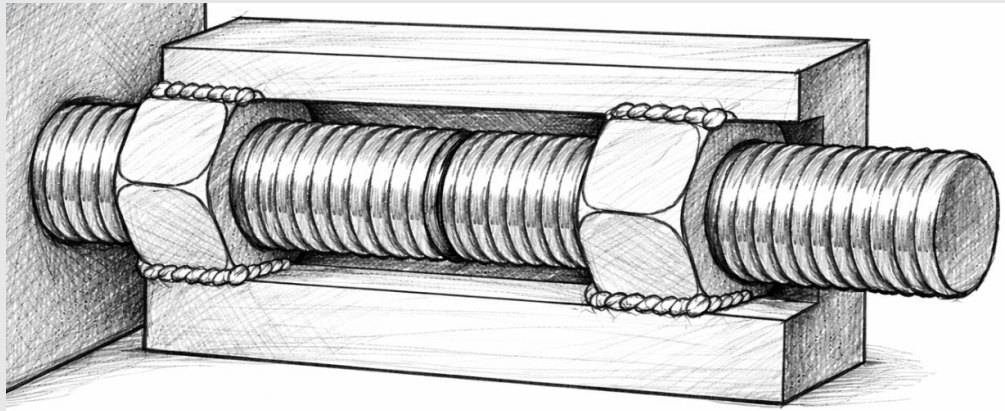
Box critical cross-section



Pin critical cross-section



# Understanding the Screw-Jack



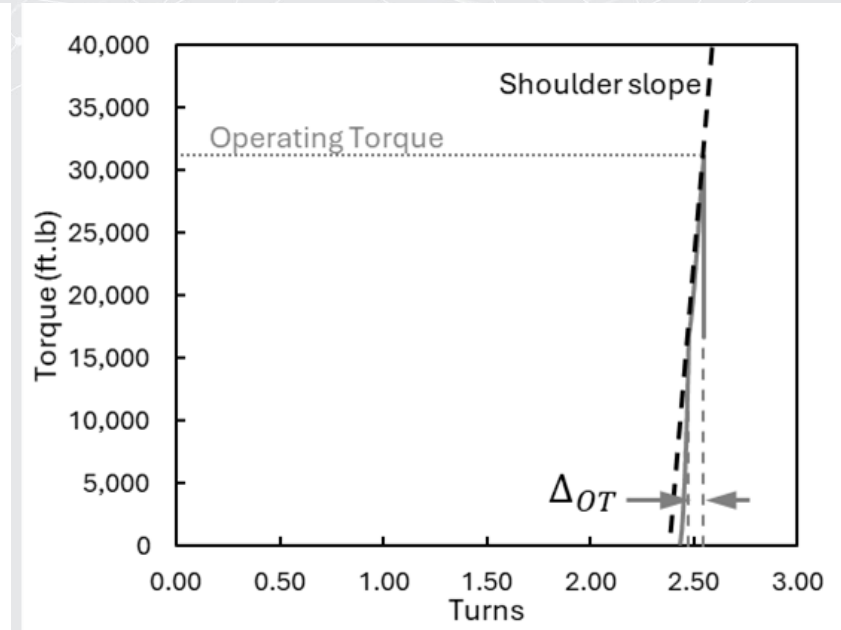
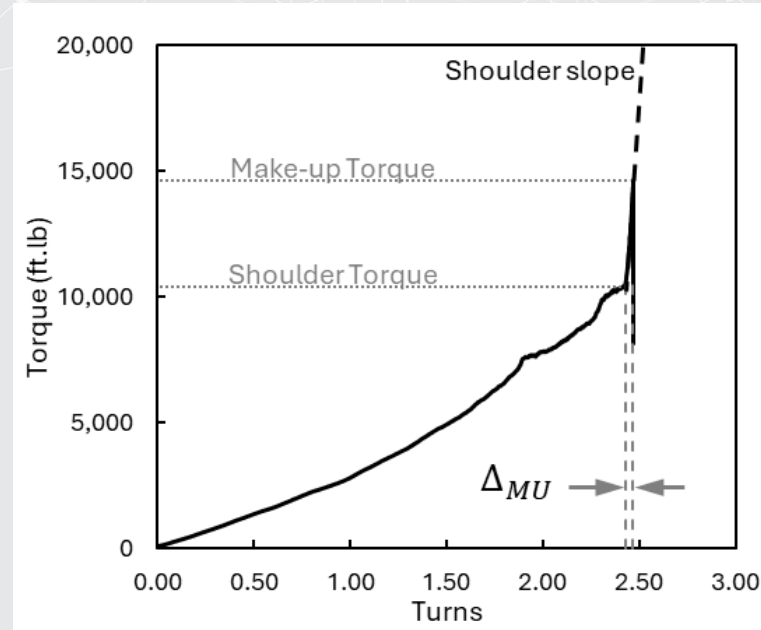
# Torque – Turn Behavior in the Field

## Example make-up:

- 5.5in 20# P110 BTC

Axial movement:

$$L_{OT} = (\Delta_{MU} + \Delta_{OT}) \times LFL$$



## Connection data sheet (CDS) values:

Make-up Torque		
Maximum	<span style="background-color: #808080; width: 20px; height: 10px; display: inline-block;"></span>	16,600 ft. lbs
Optimum	<span style="background-color: #696969; width: 20px; height: 10px; display: inline-block;"></span>	14,400 ft. lbs
Minimum	<span style="background-color: #404040; width: 20px; height: 10px; display: inline-block;"></span>	12,200 ft. lbs

Torque Limits		
Yield Torque	<span style="background-color: #FF0000; width: 20px; height: 10px; display: inline-block;"></span>	34,000 ft. lbs
Operating Torque	<span style="background-color: #000000; width: 20px; height: 10px; display: inline-block;"></span>	30,600 ft. lbs

# Torque vs Tension

## Two primary considerations

- Torsion – API RP 7G Recommended Practice for Drill String Design and Operating Limits

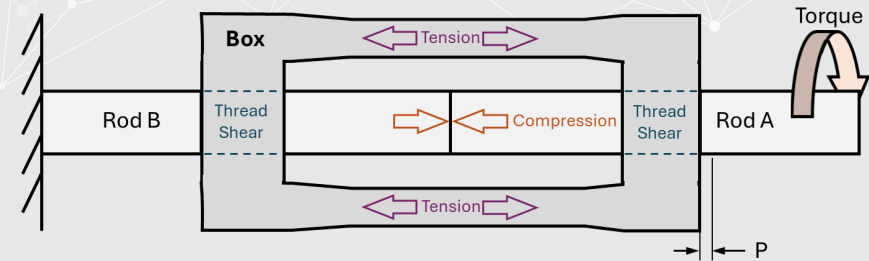
### A.8.2 TORSION AND TENSION

$$Q_T = \frac{0.096167J}{D} \sqrt{Y_m^2 - \frac{P^2}{A^2}} \quad (A.15)$$

- Screw-jack

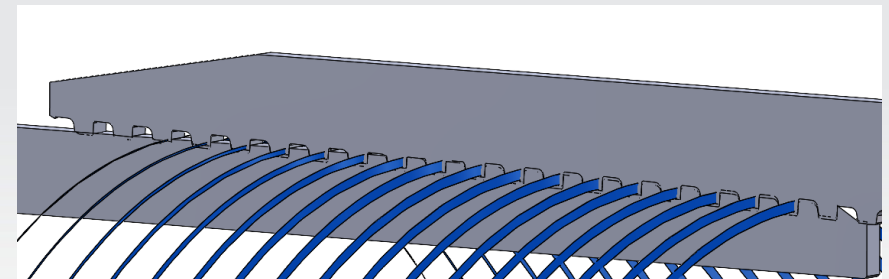
$$F_{TQ} = \frac{\delta \cdot E \cdot BCCS_{Area}}{L_B}$$

Strain (pointing to  $\delta$ )  
 Load on box critical cross section (pointing to  $F_{TQ}$ )  
 Coupling Length (pointing to  $L_B$ )



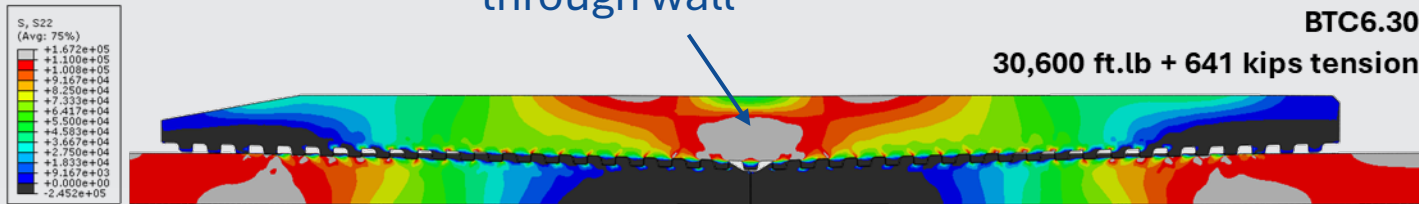
Strain component derived from axial movement resisted by:

- Pin face area in contact
- Box critical cross-sectional area ( $BCCS_{Area}$ )
- Load flank helical area



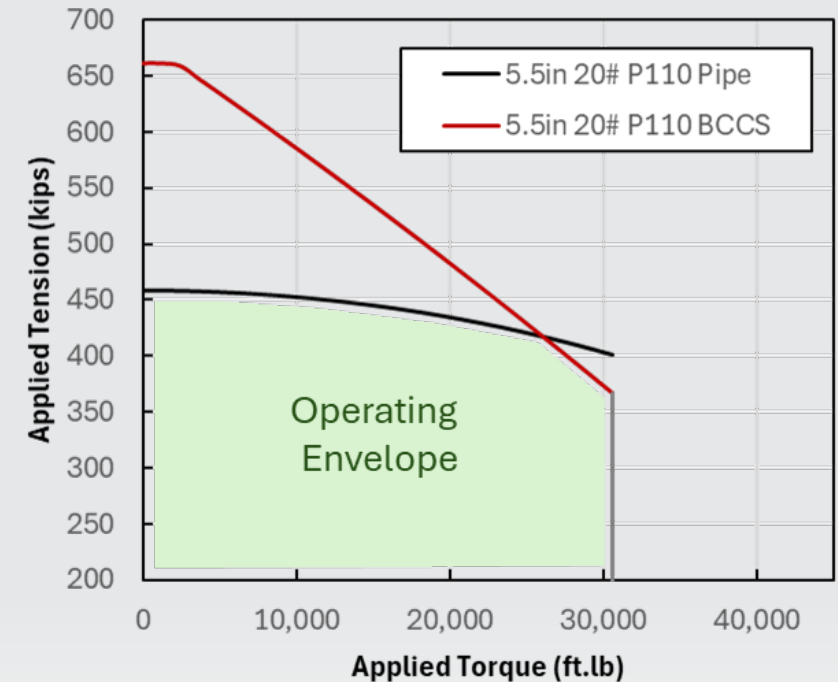
# Torque-Tension Operating Envelope

For standard sized OD – the box critical cross-section area is so much larger vs the pipe that the it only limits near the operating limits of the connection.



Axial Stresses at max rated torque + max rated tension

## BTC6.30

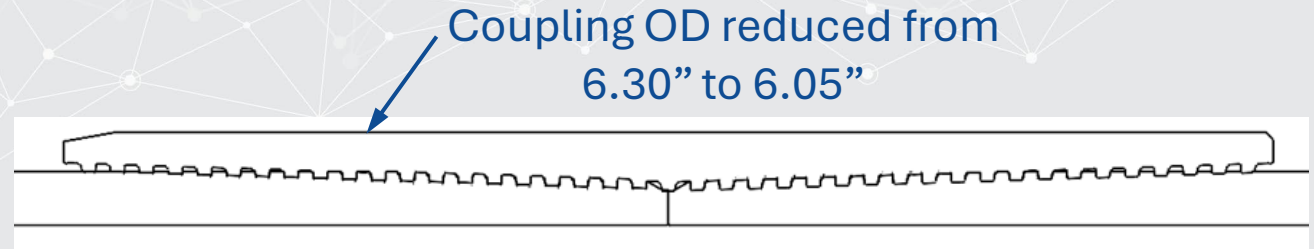


# Enhanced Clearance: Why It's Dangerous



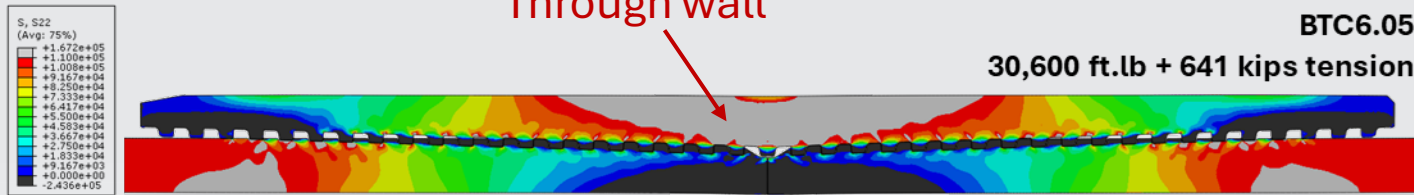
Enhanced clearance drivers

1. Improved cement
2. Regulator requirement

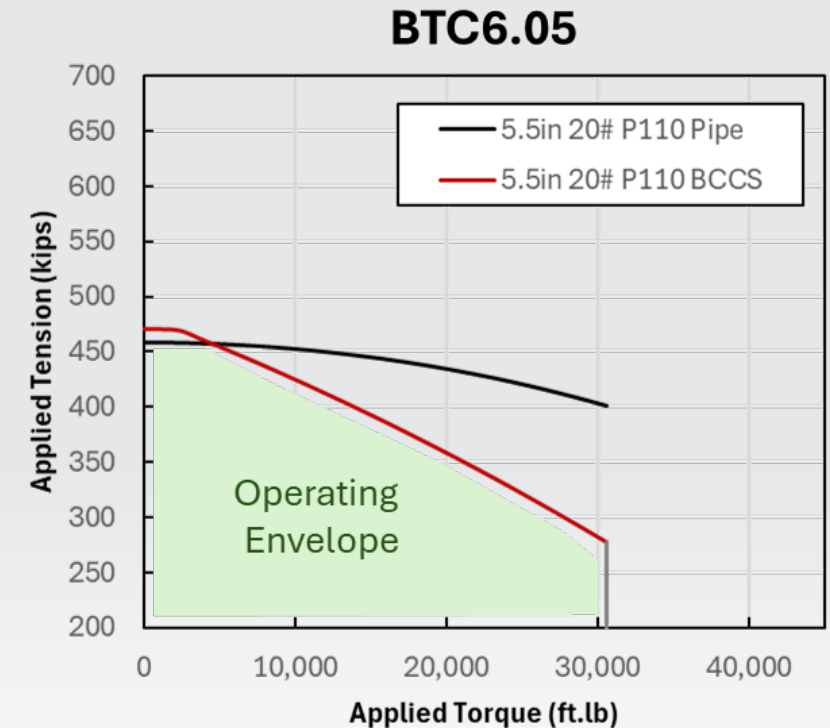


Torque-tension combined loading

- The box becomes the weak-link at even low torques
- Even with tension safety factors a coupling has high risk of failure



Axial Stresses at max rated torque + max rated tension



# Field Example



## Case

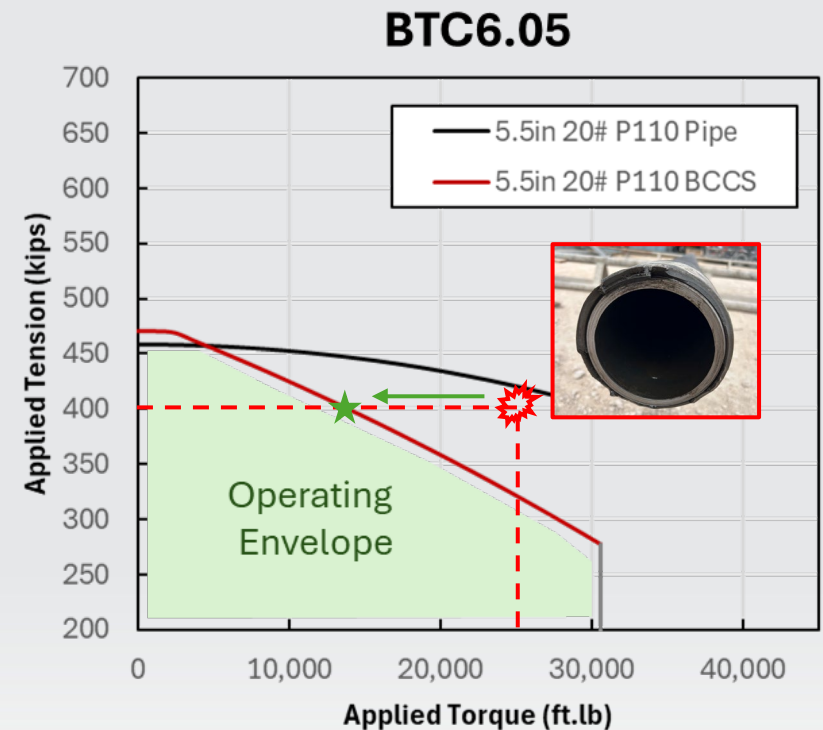
- 5.5in production casing with enhanced clearance run inside of 7-5/8in intermediate casing.
- 4-mile laterals (10K ft TVD)

## Problem

- High torque was required to reach bottom.
- Coupling parted near surface.

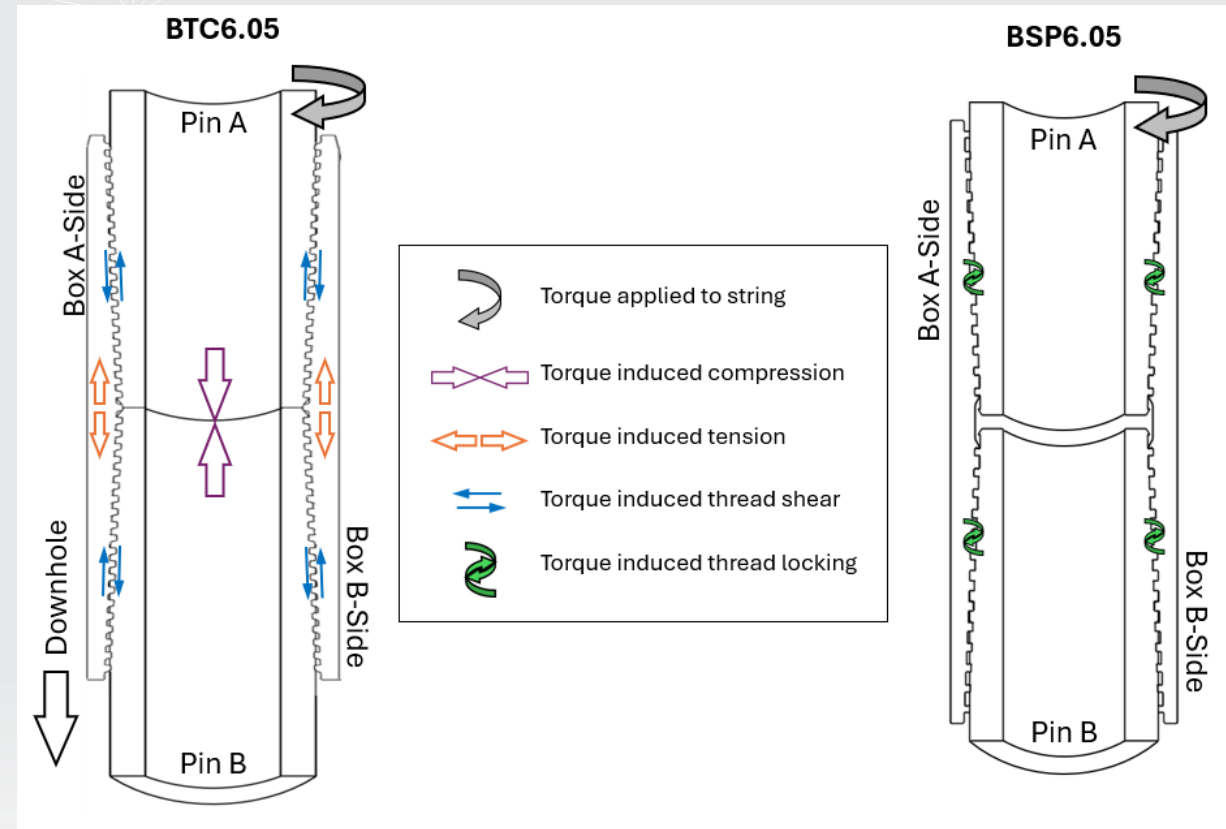
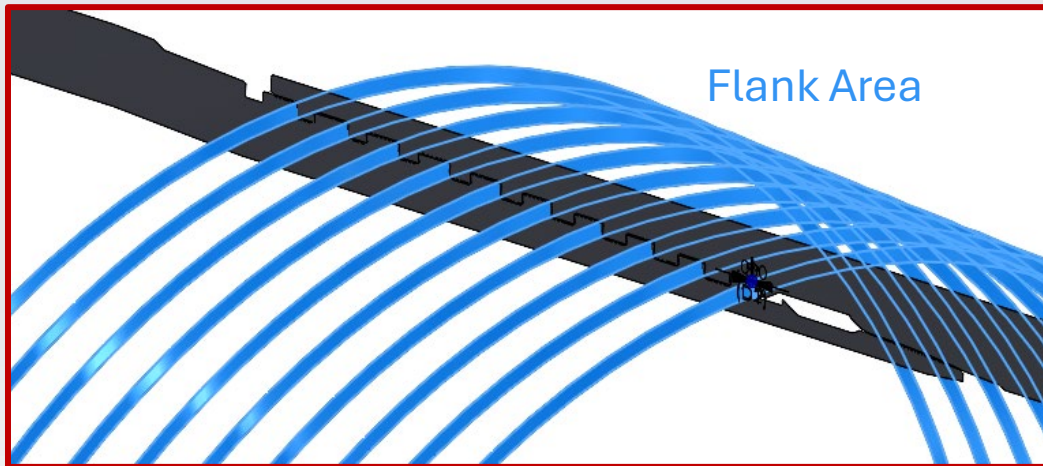
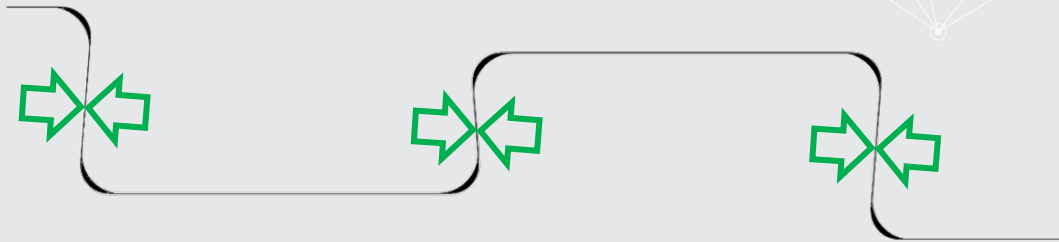
## Mitigation

- Utilize operating envelope for remainder of casing
- Move to an alternate casing connection



# Wedge Threads: Different Physics

How to balance torque / tension / clearance

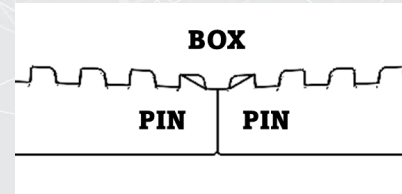
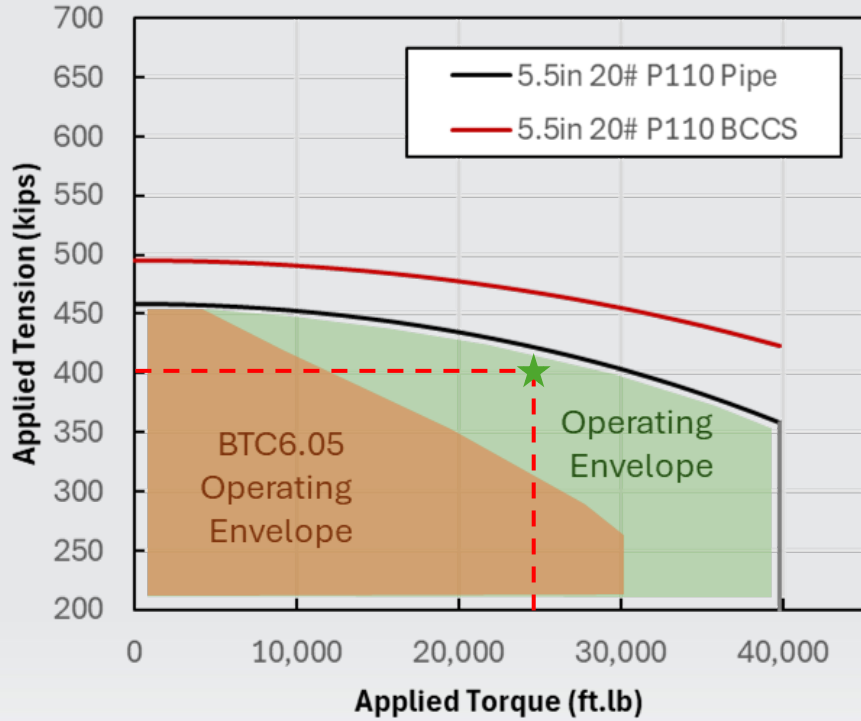


Proprietary wedge-thread example  
(BSP6.05)

# Wedge Threads vs Shouldered

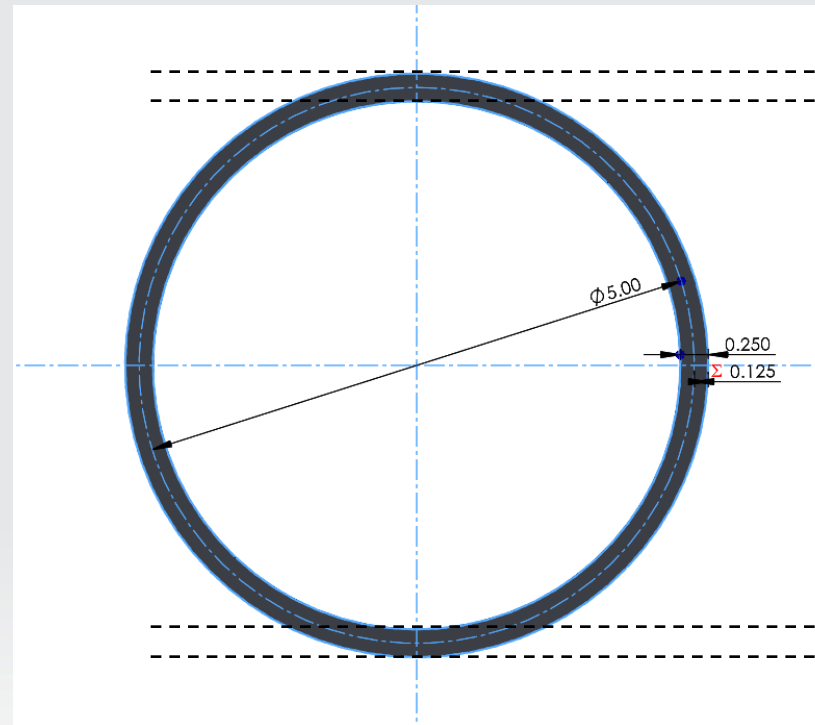


## BSP6.05



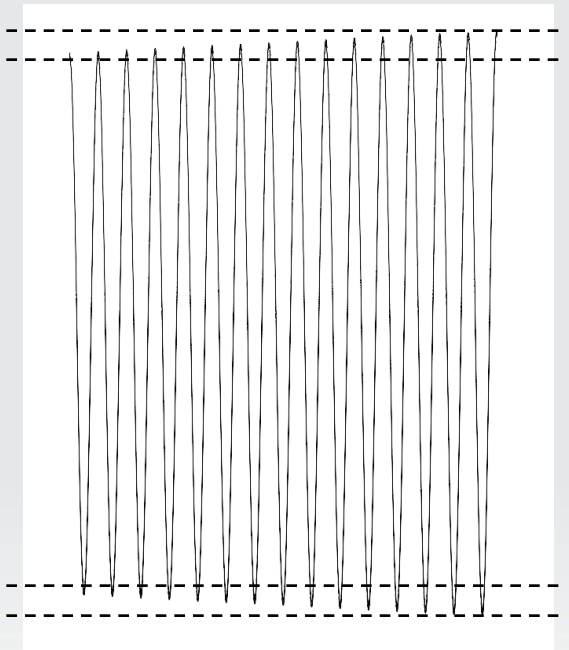
Over the same wall the stab flank area is **3 times larger** than the shoulder area

### Shouldered Connection



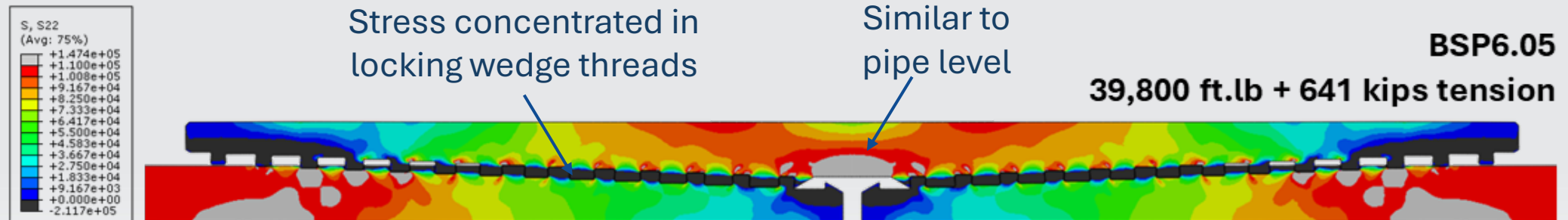
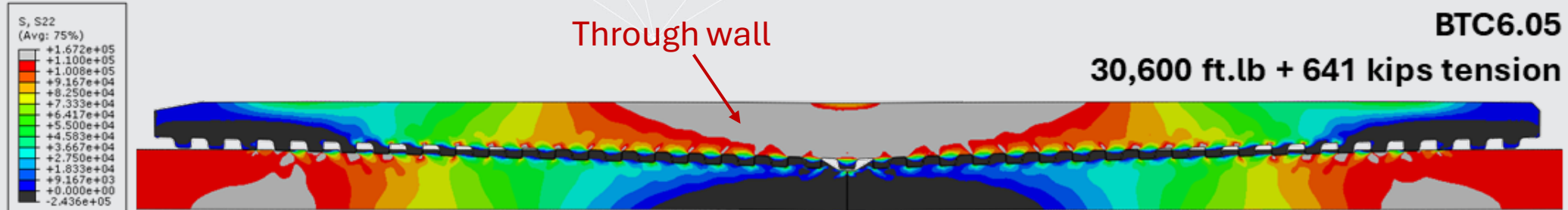
Shoulder area = 3.9 in<sup>2</sup>

### Wedge



Flank area = 12.0 in<sup>2</sup>

# Wedge vs Shouldered

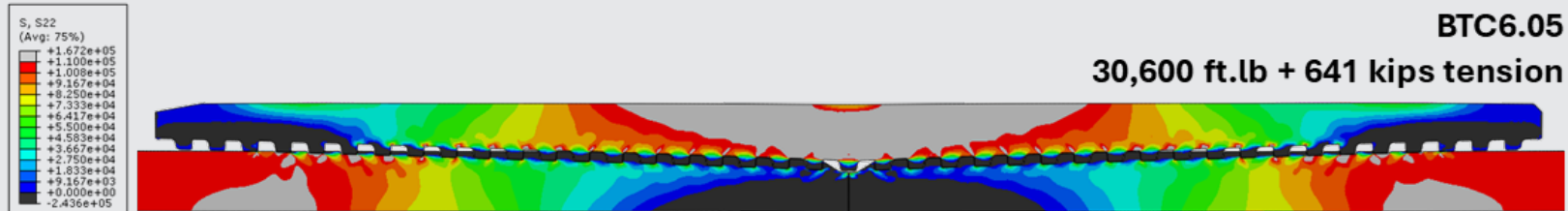


# Regulatory Challenge Example

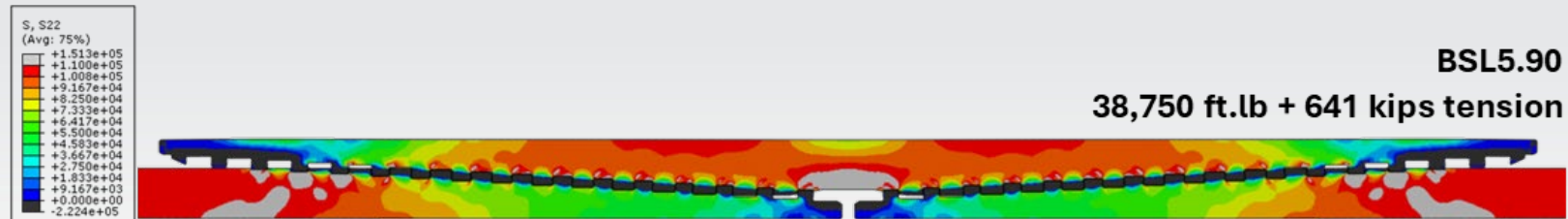
Running 5.5in 20# pipe inside of 7.625in 29.7#

Max coupling OD of 5.9in to meet the 0.422in radial clearance required in BLM 43 FR Part 3160

- Slim-line version of BTC would be even more restrictive



- Slim-line version of the wedge connection has no additional restrictions

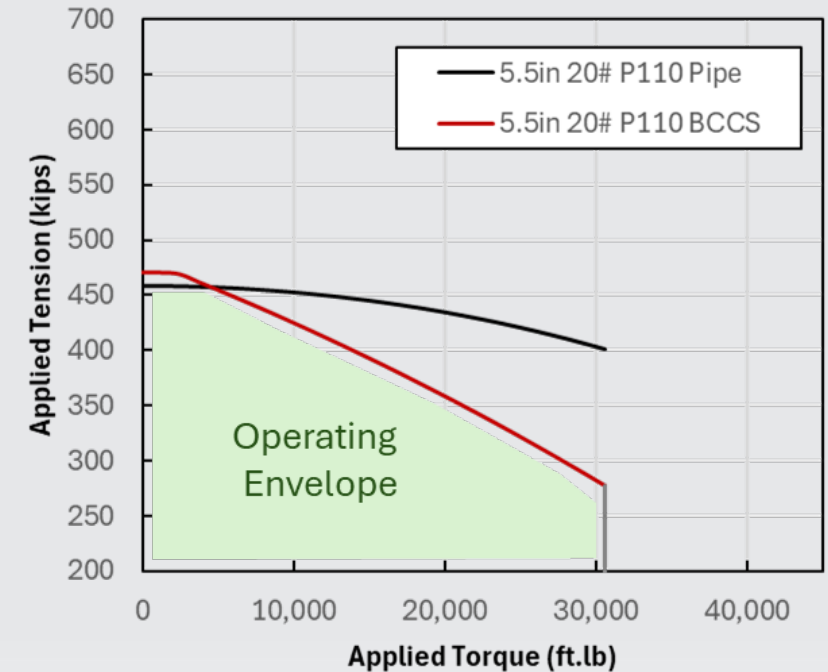


# Key Takeaways



- Top drives are no longer the limiting factor for ERD
- Know the torque-tension envelope for the connection you are running
  - Do not rely solely on Connection Data Sheet ratings
- Match the right connection for the application
  - If you can not compromise - move to a connection that does not have the screw jack mechanism

## Torque Tension Envelope



# Questions?

**THANK YOU**